

FUN3D Solutions for Tandem Cylinders

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Outline



- Objectives
- Numerical Method
- Flow Conditions
- Grids
- Results
- Computational Resources
- Observations

Objectives



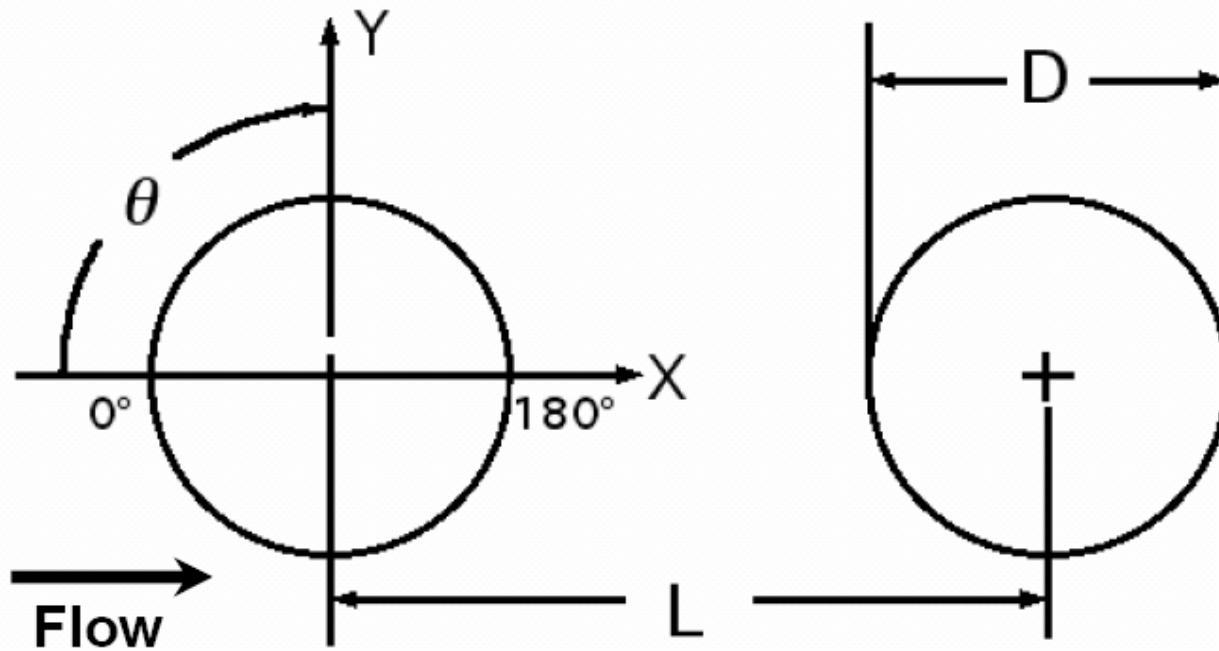
- Assess the applicability of an unstructured grid flow solver, FUN3D, for Tandem Cylinder Configurations
- Examine turbulence modeling sensitivity



Numerical Method

- Equations solved
 - Unsteady Reynolds-averaged Navier-Stokes (URANS) equations
Fully unstructured node-based flow solver (FUN3D)
 - Modified Menter's two-equation shear stress transport (SST) model
(Ref. Khorrami et al. AIAA-2006-3203)
 - Hybrid RANS/LES model (Ref. Lynch et al. AIAA-2008-3854)
 - MDDES hybrid RANS model (Ref. Vatsa AIAA-2010-4001)
- Spatial and temporal discretizations
 - Roe's flux-difference splitting scheme without flux limiter
 - Second order spatial accuracy
 - Optimized second-order backward difference (BDF2OPT) scheme for temporal discretization
- Boundary Conditions
 - Adiabatic walls, Riemann invariants at far-field, spanwise periodicity

Flow Conditions in Simulations

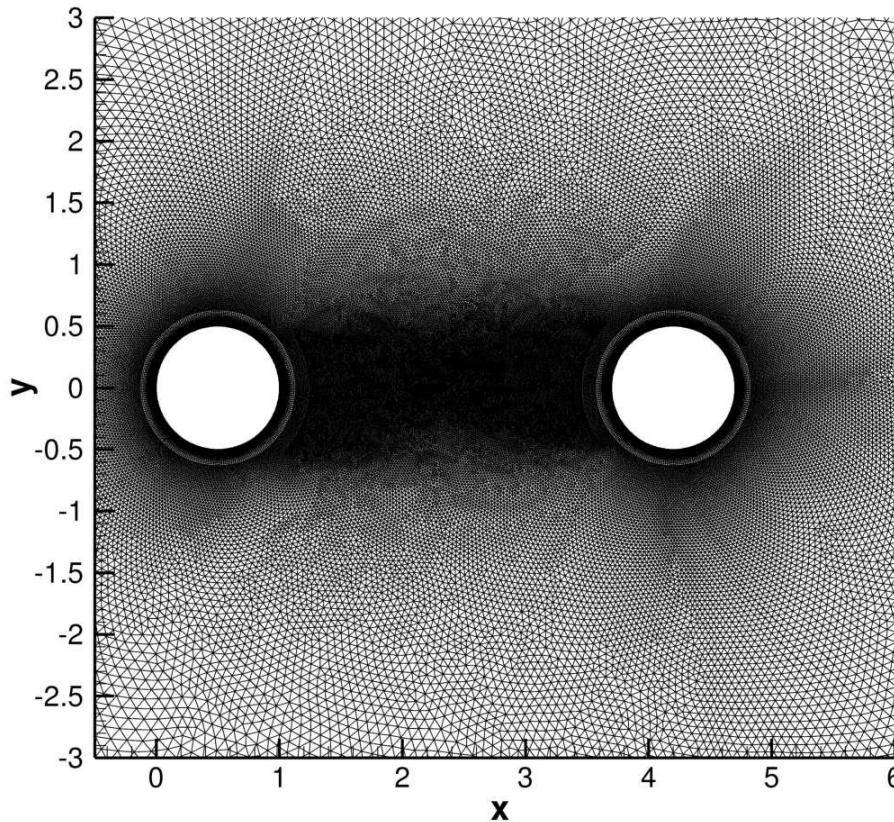


- $Re = 166,000$ based on D
 - Flow code run in fully turbulent mode
 - Surface Transition occurs approx. at $\theta = 60\text{-}70$ deg.
- $M = 0.166$

Computational grid (partial view)



- Grid type: Structured near solid surfaces, unstructured away from boundary-layer regions
- Grid size: 8.7 million nodes, 97 spanwise planes
- Extent of grid: Far-field 25 D in x-y plane, span of 3 D





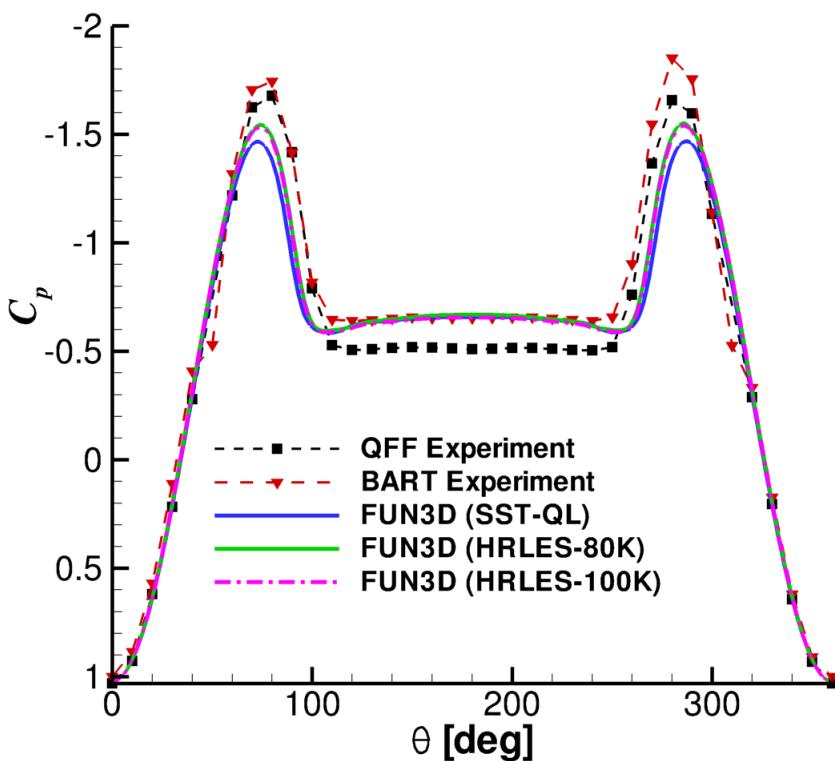
Results

- Time step
 - 1.69×10^{-5} seconds
- Number of time steps run
 - Total : 80,000 – 100,000 time steps
 - Sampling : Minimum of 50,000 time steps
- Sheding frequency
 - 160 Hz
- Time-averaged Drag (C_d per unit span)
 - 0.637 on front, and 0.446 on rear cylinder for SST-QL
 - 0.436 on front, and 0.207 on rear cylinder for “MDDES other solution”
- Convergence information
 - C_p and $C_{p_{rms}}$ checked after every 10,000 time steps

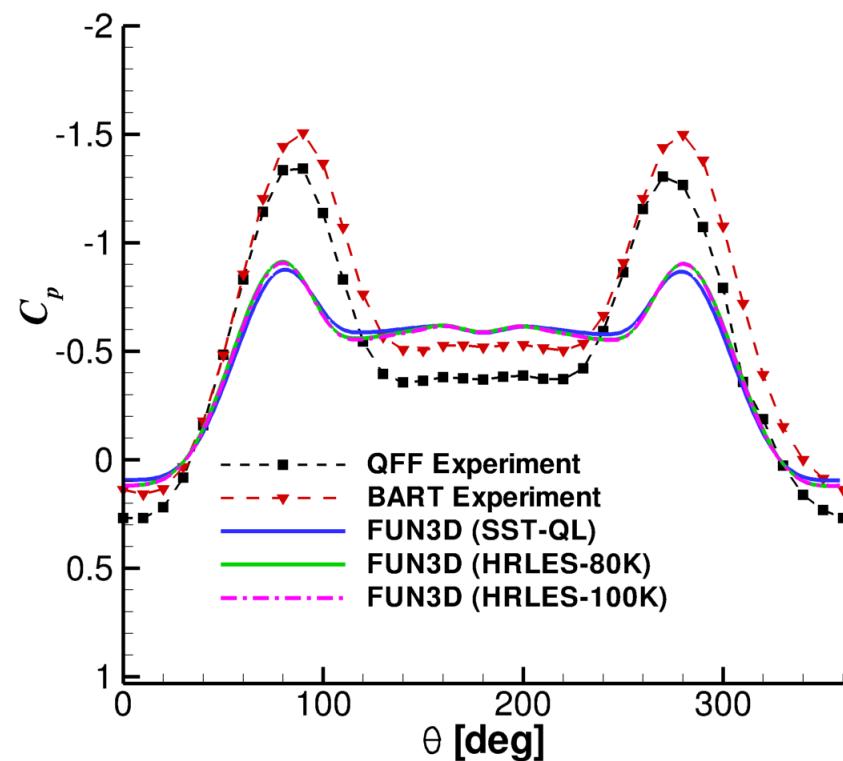
Surface Pressure



- Collect time-average surface pressure data after initial transients exit computational domain
- Average the surface data along span for given θ



Front cylinder

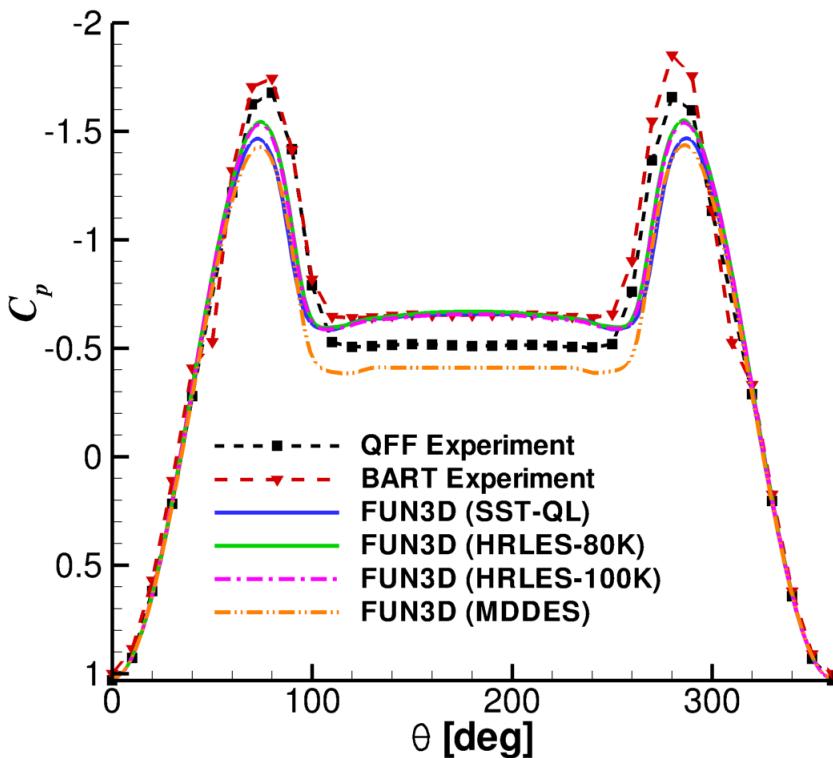


Rear cylinder

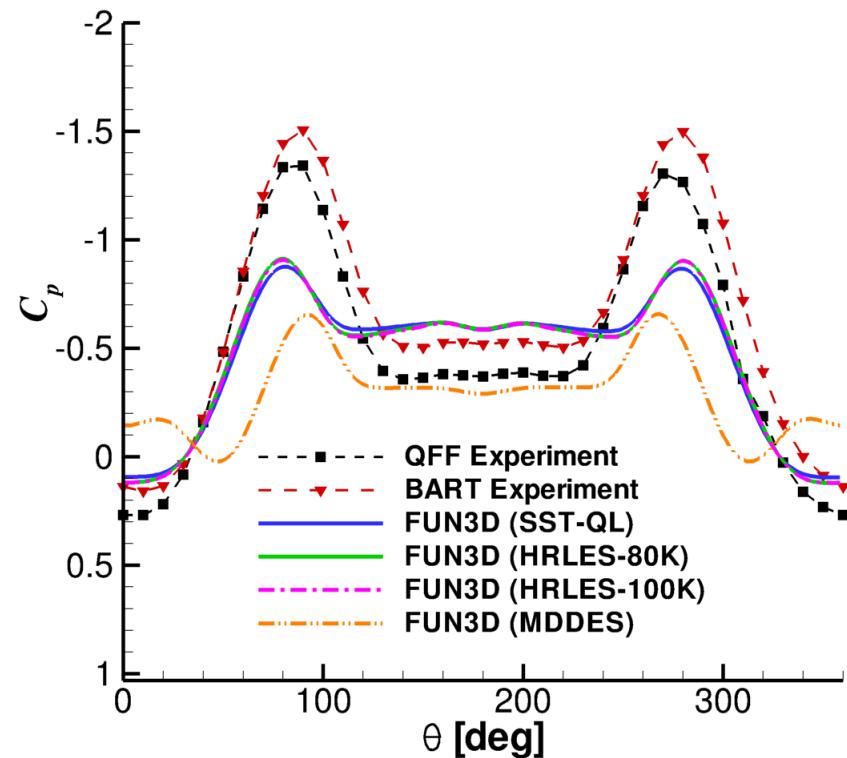
Surface Pressure



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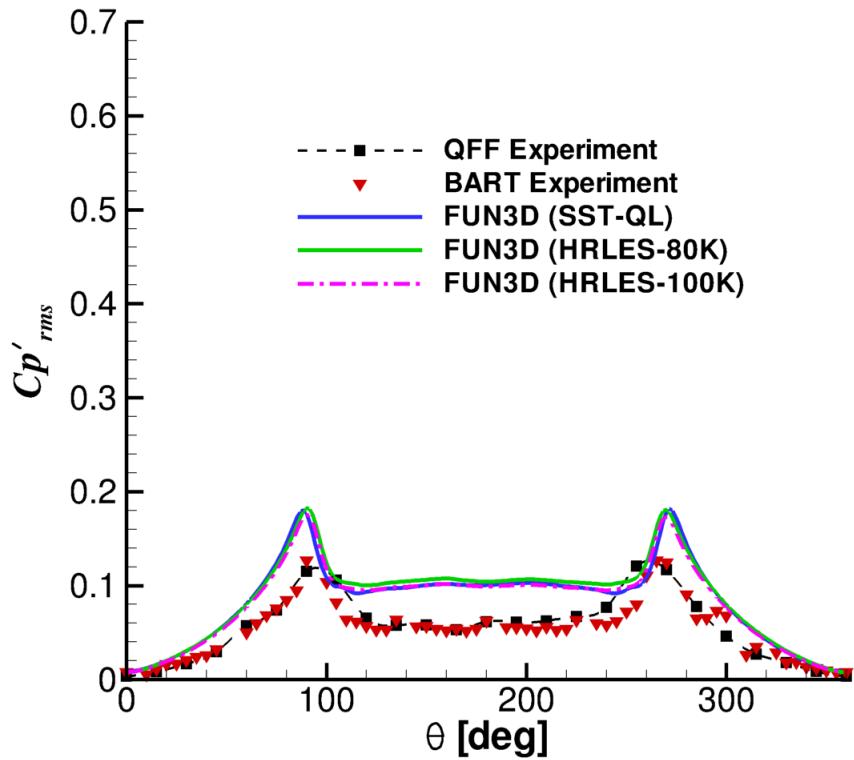


Front cylinder

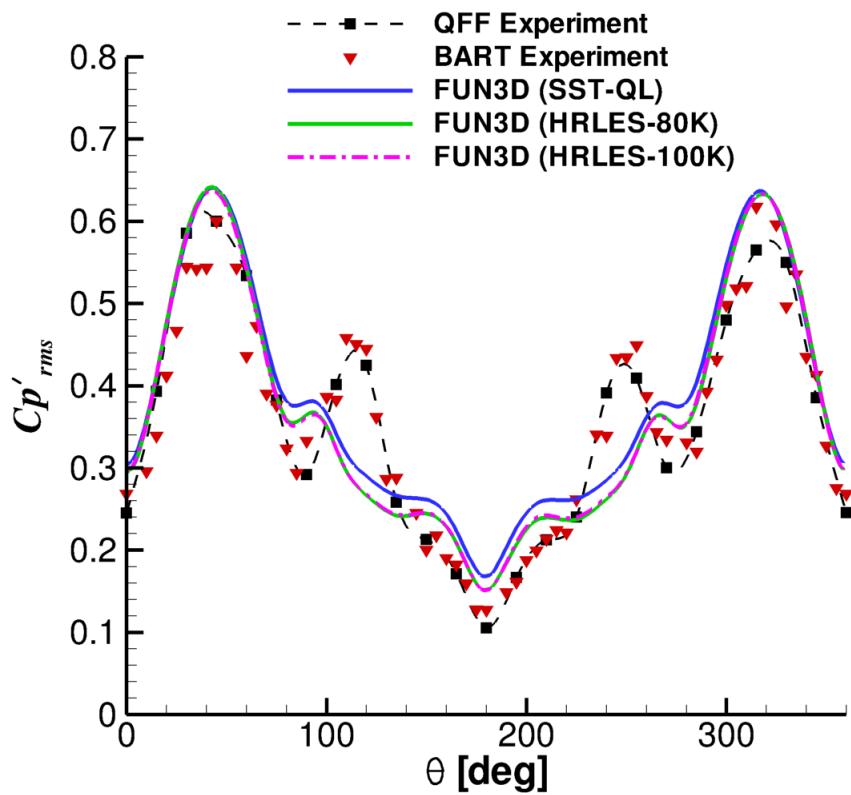


Rear cylinder

RMS of Surface Pressure

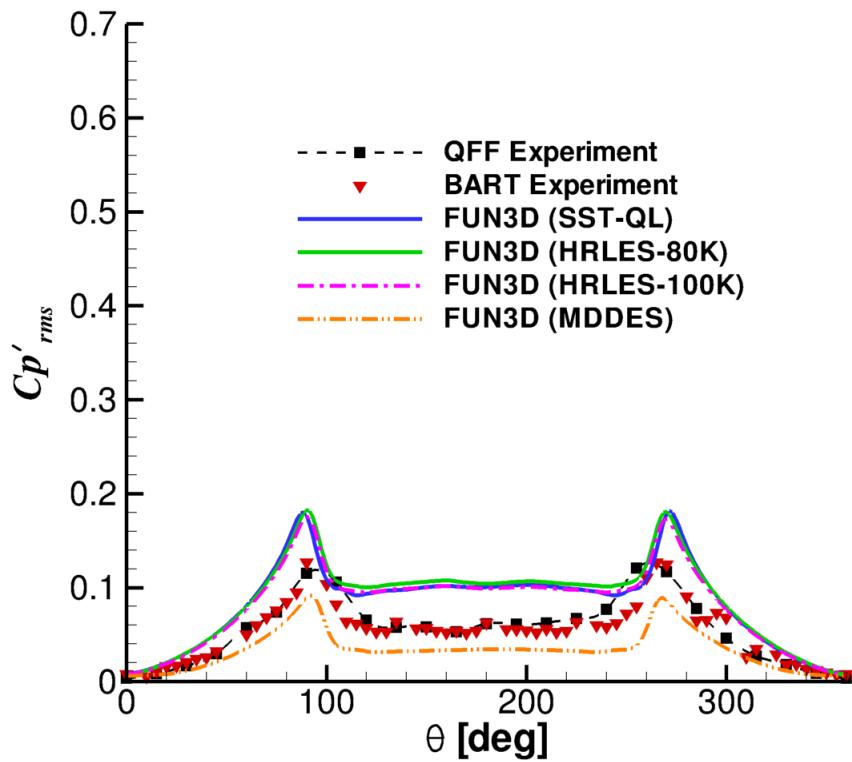


Front cylinder

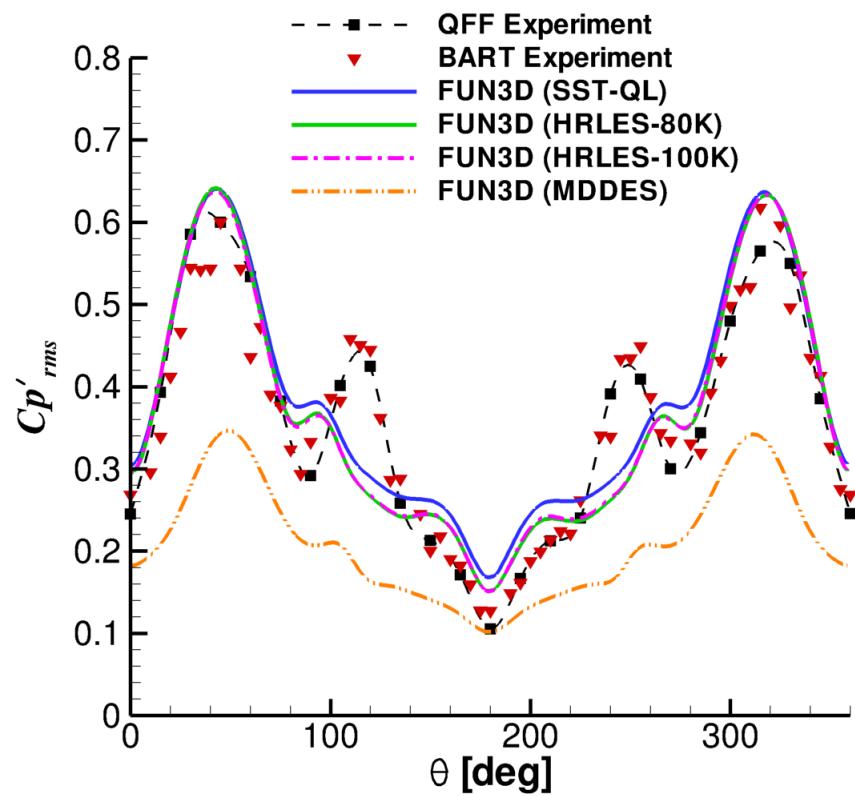


Rear cylinder

RMS of Surface Pressure

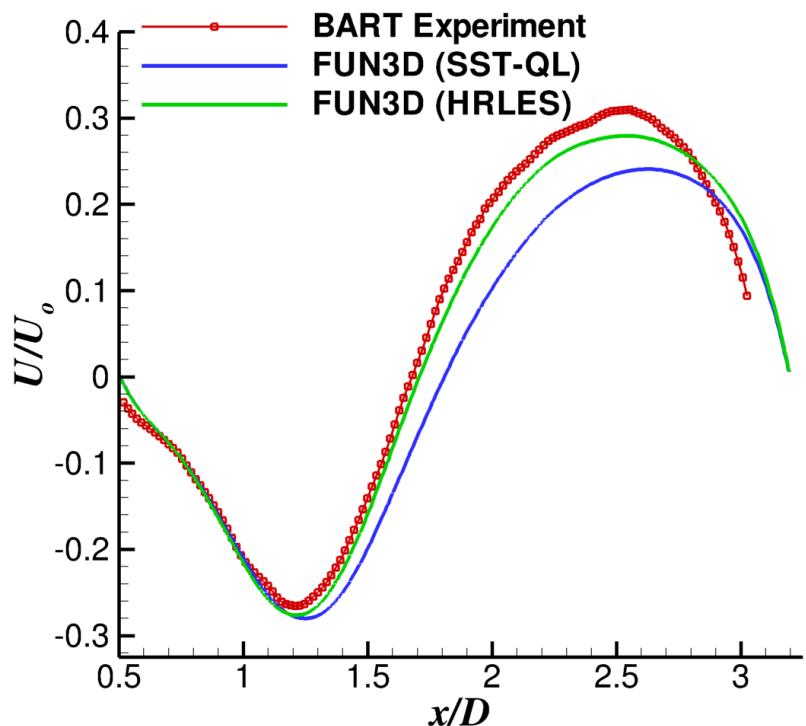
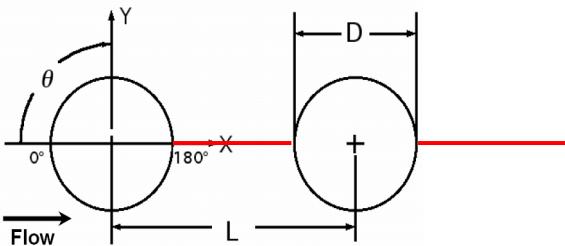


Front cylinder

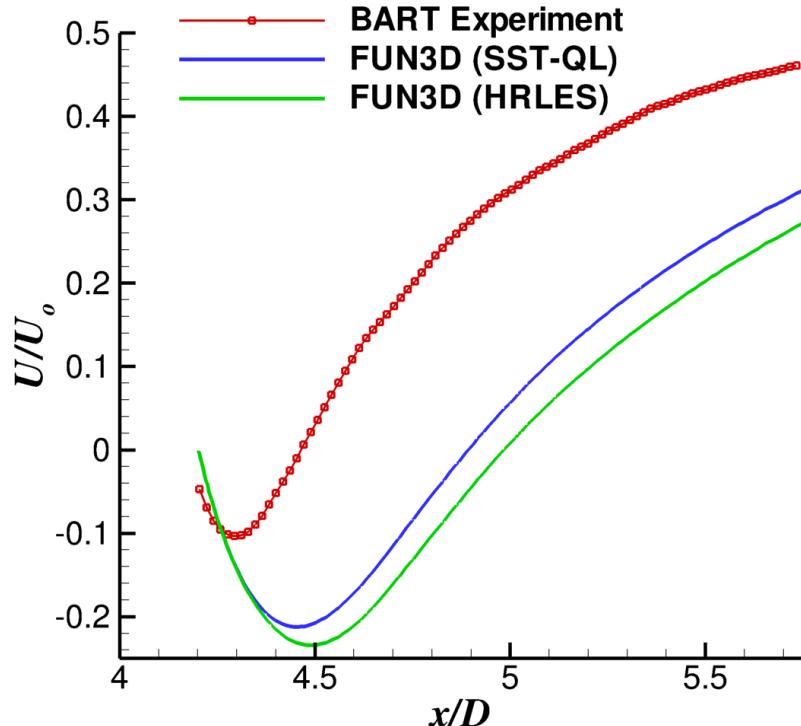


Rear cylinder

Mean Velocity along $y/D=0$

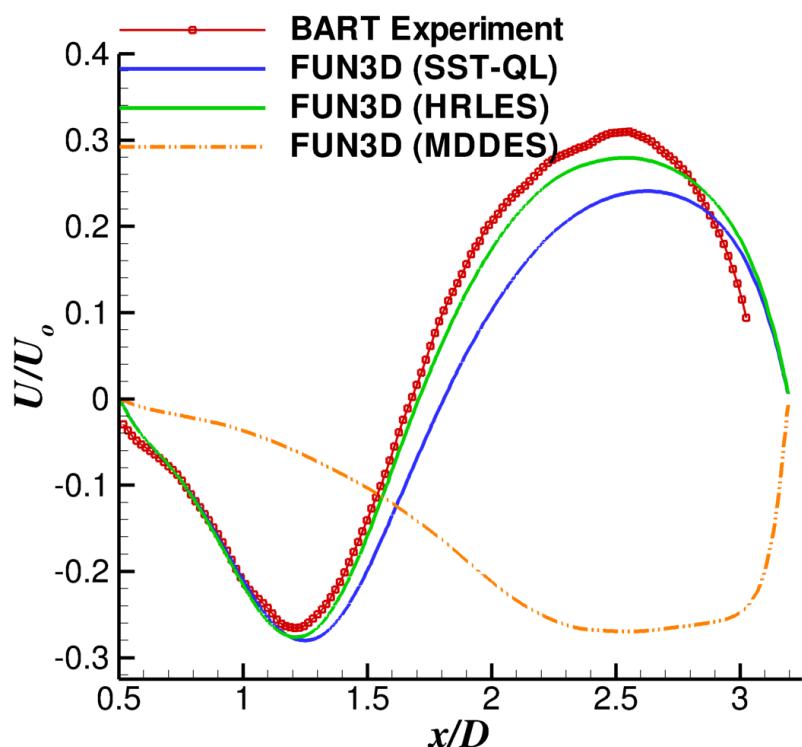
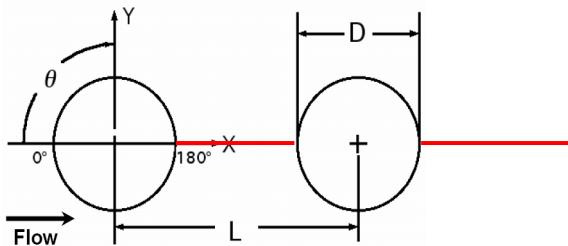


Gap Region

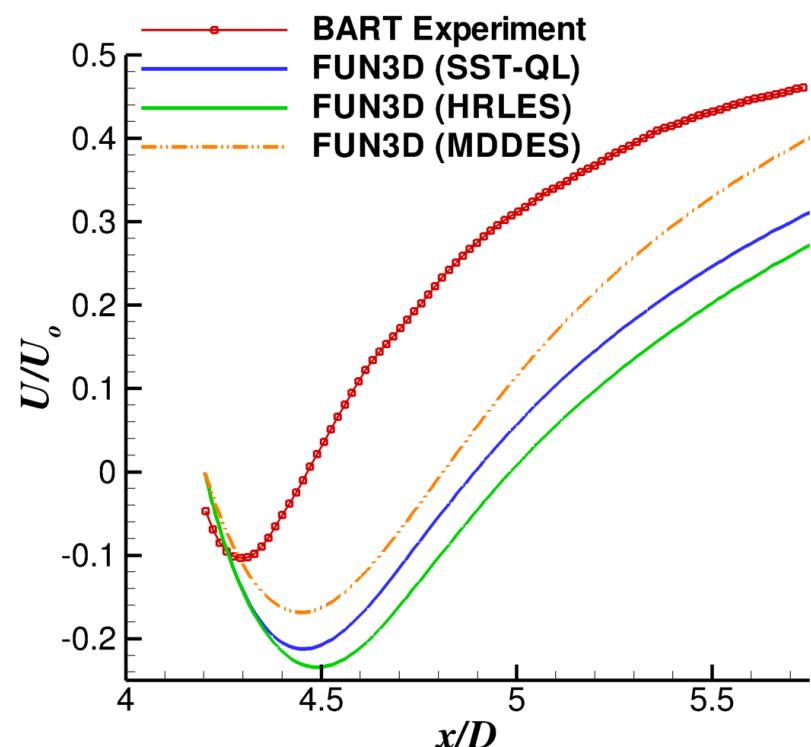


Aft of Downstream
Cylinder

Mean Velocity along $y/D=0$



Gap Region

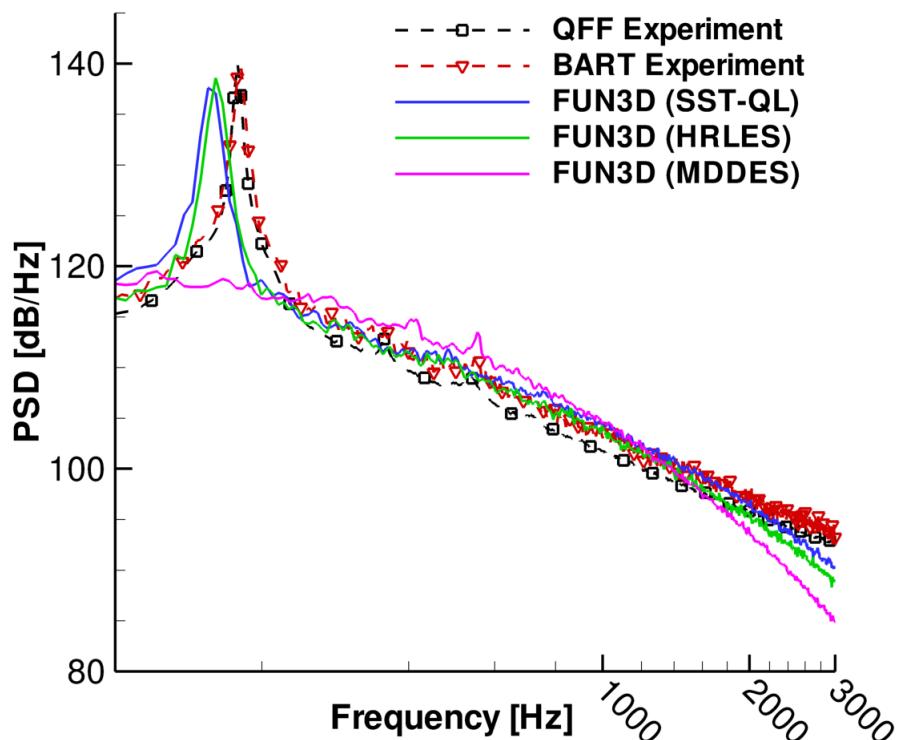
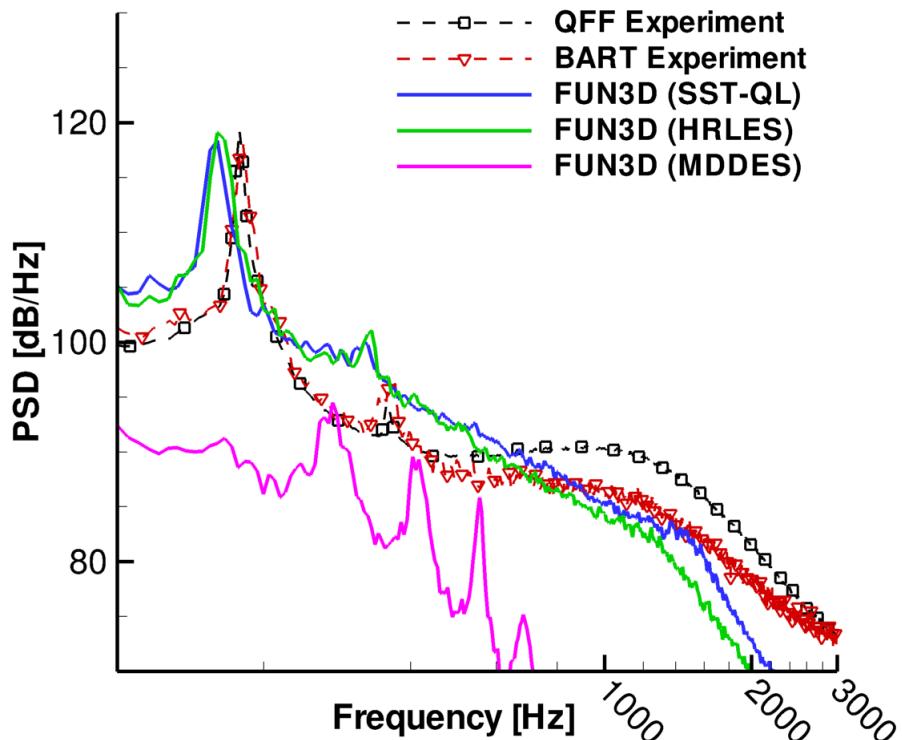


Aft of Downstream Cylinder

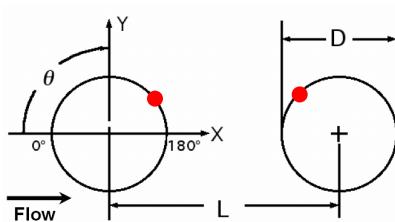
Surface Pressure Spectra



- Power Spectral Density



Upstream, $\theta = 135^\circ$



Downstream $\theta = 45^\circ$

Computational Resources



- Computer hardware
 - CPU: 128 nodes/256 cores of Wolfdale 3 GHz single cpu/dual core cluster
 - Interconnect: Gigabit ethernet
- Resources
 - CPU (or wall clock) Time / time step : 32.4 secs.
 - 80,000 time steps in simulation
 - CPU (or wall clock) Time / 1 sec of simulation time: 532 hrs.
 - 59,200 time steps needed for 1 sec of simulation time
 - Memory used
 - Per cpu: 550 MBytes
 - Total: 70.4 GBytes

Observations



- What did you learn?
 - Computational challenges
 - Significant computational effort for statistically meaningful results
 - Constructing suitable grids very challenging
 - New insights into the physics
 - Transition difficult to simulate, large sensitivity to transition location
 - Assessment of state-of-the-art based on your simulation for the problem category of interest
 - Encouraging results with unstructured grid flow code FUN3D, solutions capture salient flow features
 - Uncertainty due to grids, transition and turbulence modeling
 - Benchmark deficiencies
 - Reynolds number too low, transition sensitive
 - Recommendations for follow-on efforts
 - Additional measurements: Need high Reynolds number tests
 - Procedures for computations or measurements: Need systematic grid refinement studies, better turbulence/transition modeling

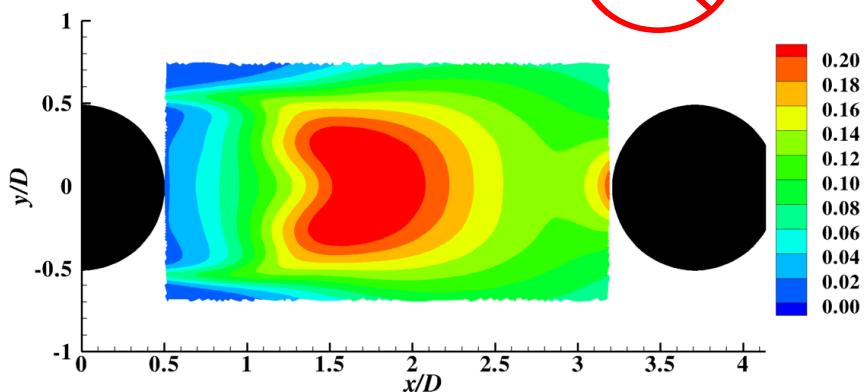


OPTIONAL SLIDES

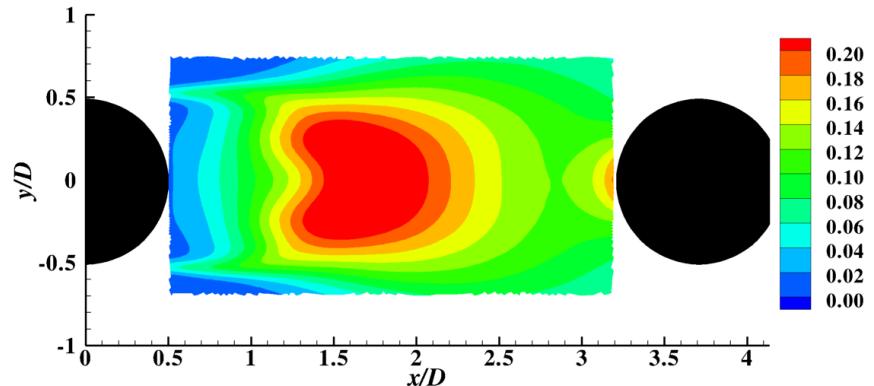
2-D TKE contours in gap region



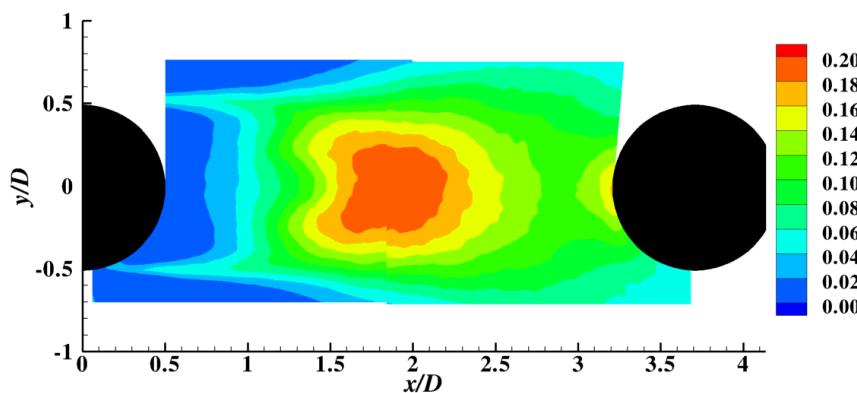
• $\frac{1}{2} (u' u' + v' v' + \cancel{w' w'}) / U_o^2$



FUN3D (SST-QL)

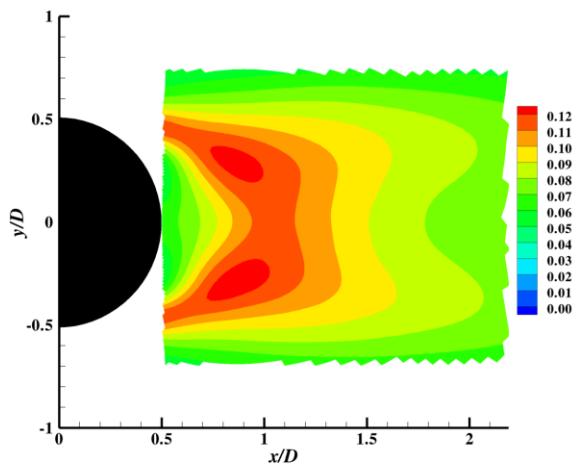


FUN3D (HRLES)

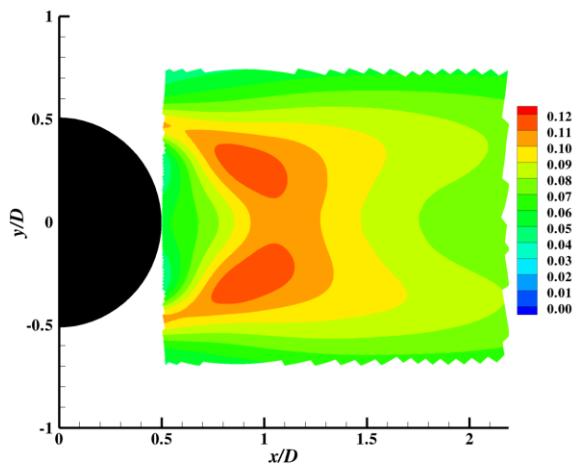


Exp. PIV Data (BART)

2-D TKE contours aft of rear cylinder

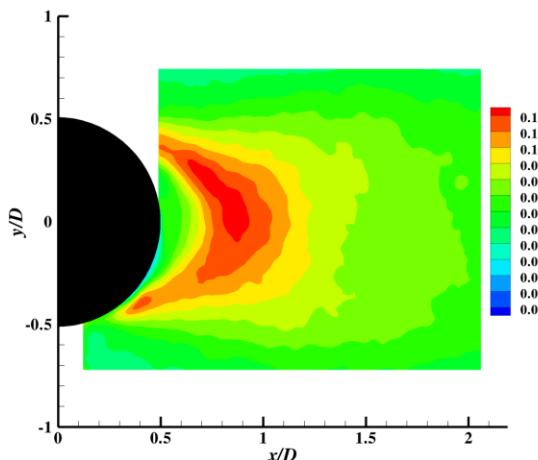


FUN3D (SST-QL)



FUN3D (HRLES)

- $1/2 (u' u' + v' v' + \cancel{w' w'}) / U_o^2$

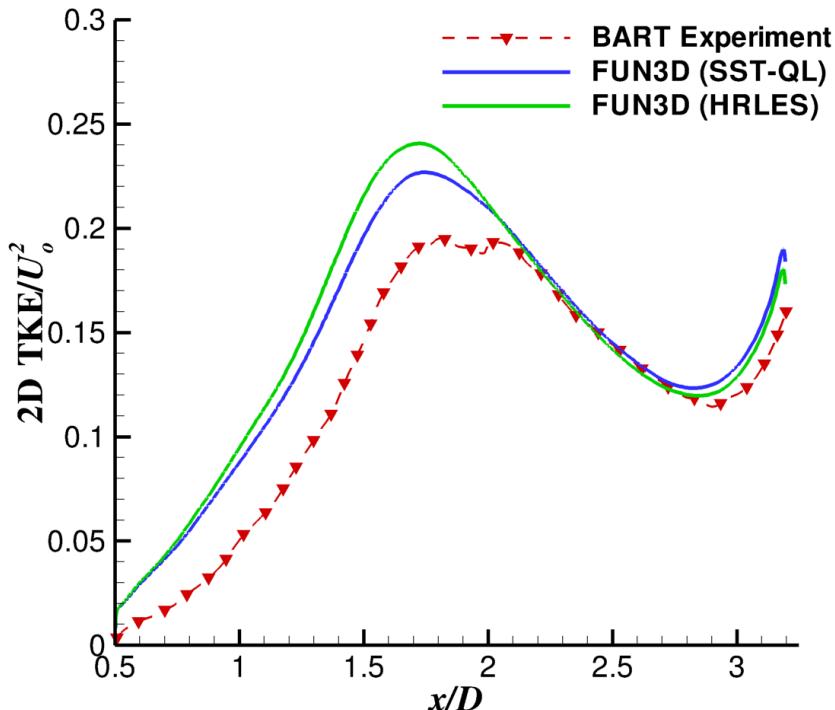


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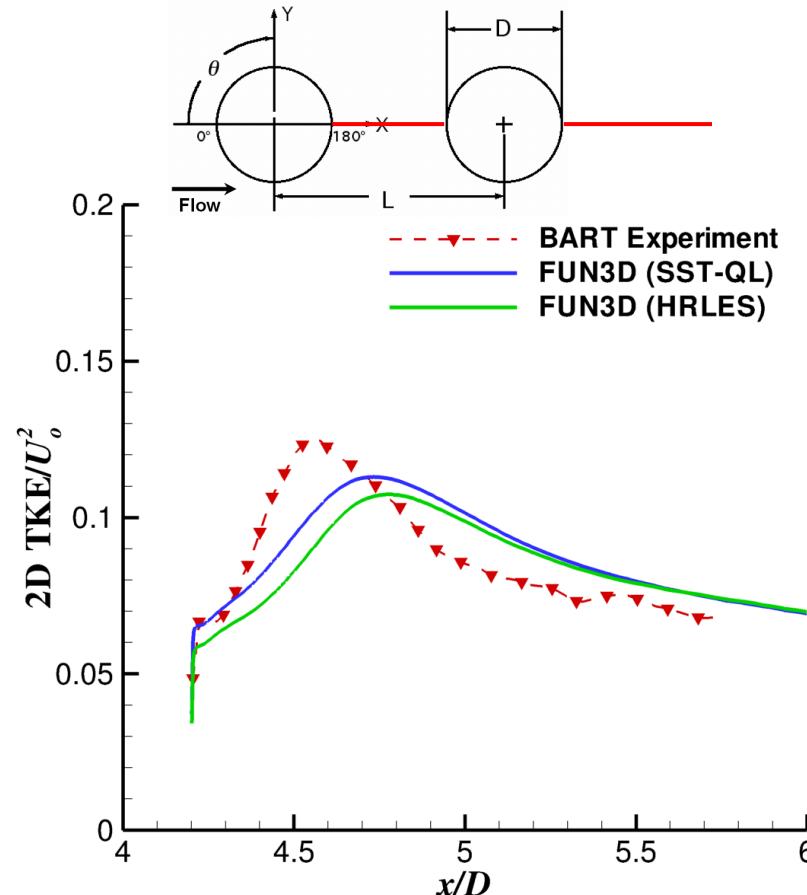
2D TKE comparisons along $y/D=0$



- $\frac{1}{2} (u' u' + v' v' + \cancel{w' w'}) / U_0^2$



Gap region between cylinders

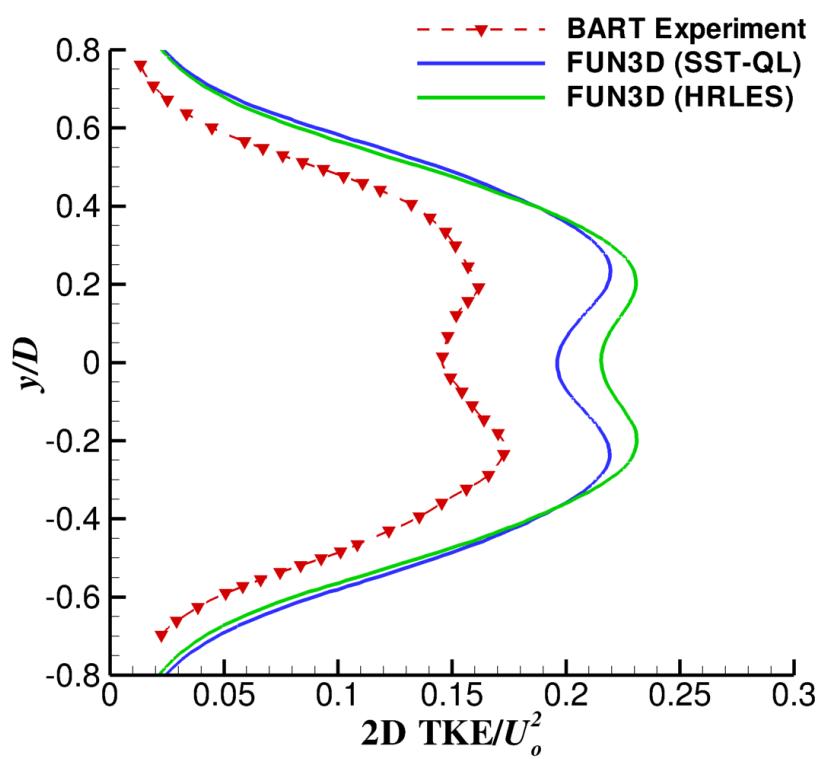


Aft region of rear cylinder

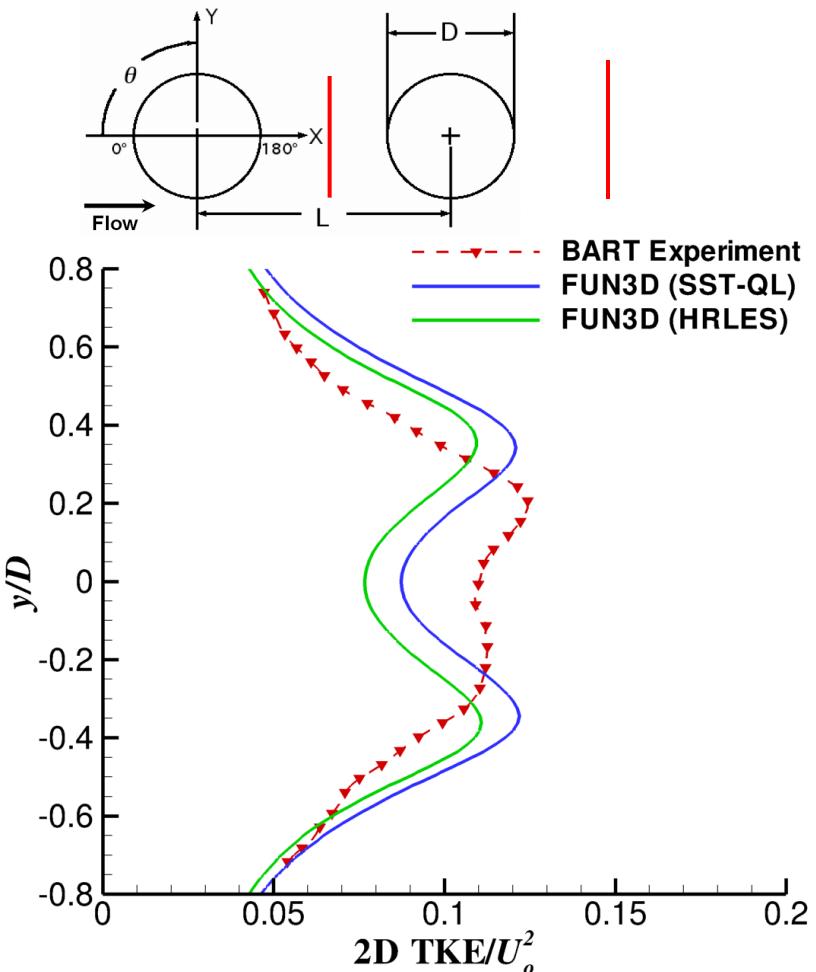
2D TKE comparisons at fixed x locations



- $\frac{1}{2} (u' u' + v' v' + \cancel{w' w'}) / U_o^2$



$x = 1.5 D$, Gap region



$x = 4.45 D$, Aft region